

DYNAMIC MODELLING AND OPTIMIZED MODEL PREDICTIVE CONTROL STRATEGIES FOR THE ORGANIC RANKINE CYCLE

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Introduction

- ✓ Organic Rankine Cycle (ORC) : Small scale, lower temperature
- ✓ Application fields : Waste heat recovery, Solar sources, engine exhaust gases, biomass CHP, geothermy, ...
- Selection of the working fluid and optimization of the working conditions are key issues in waste heat recovery applications
- Dynamic models are required to define proper control strategies under transient conditions



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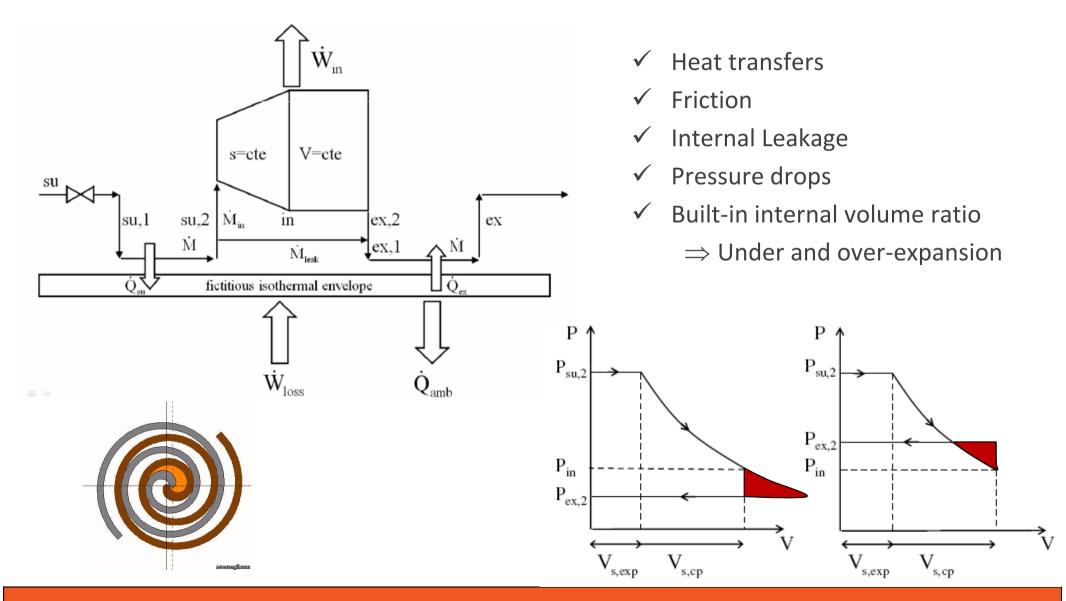
Introduction

Steady-state models

- Description
- Experimental validation
- Dynamic models
- Cycle optimization and control
- Model predictive control
- Conclusions



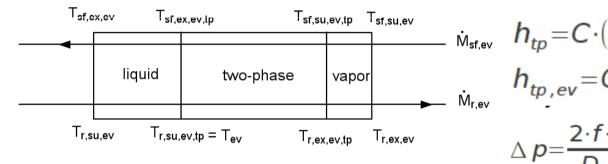
Volumetric expander model





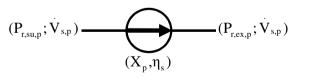
ORC cycle steady-state model

Plate Heat exchangers model:

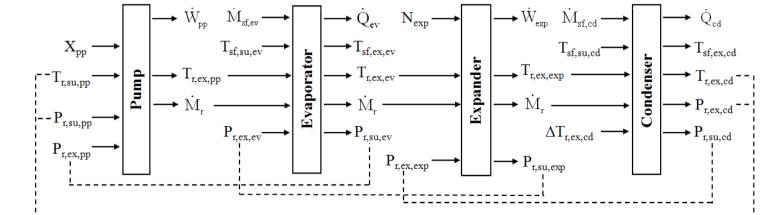


 $Nu = C \cdot Re^{n} \cdot Pr^{n}$ $= \dot{h}_{tp} = C \cdot (0.25 \cdot Co^{-0.45} \cdot FR_{I}^{0.25} + 75 \cdot Bo^{0.75})$ $= \dot{h}_{tp,ev} = C \cdot h_{I} \cdot Bo^{0.5}$ $= \dot{h}_{tp,ev} = \frac{2 \cdot f \cdot G^{2}}{D_{h} \cdot \rho} \cdot L$

Pump model:

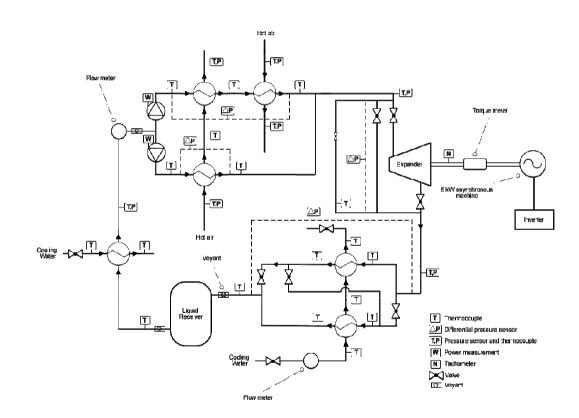


Cycle model:





Experimental validation



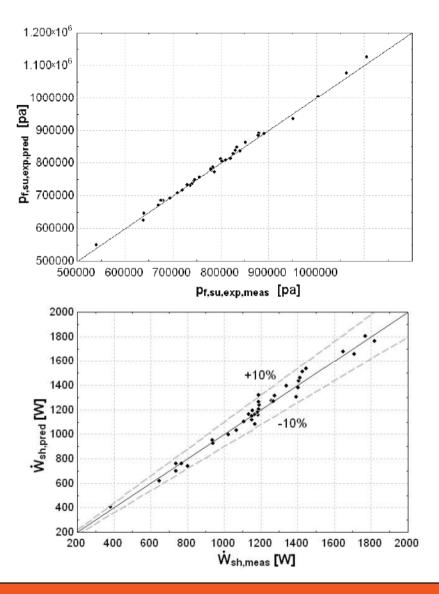




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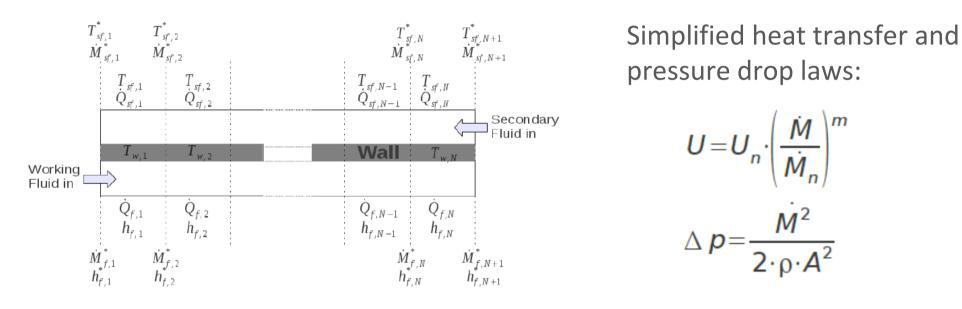
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HEAT EXCHANGER MODEL



Conservation of energy:

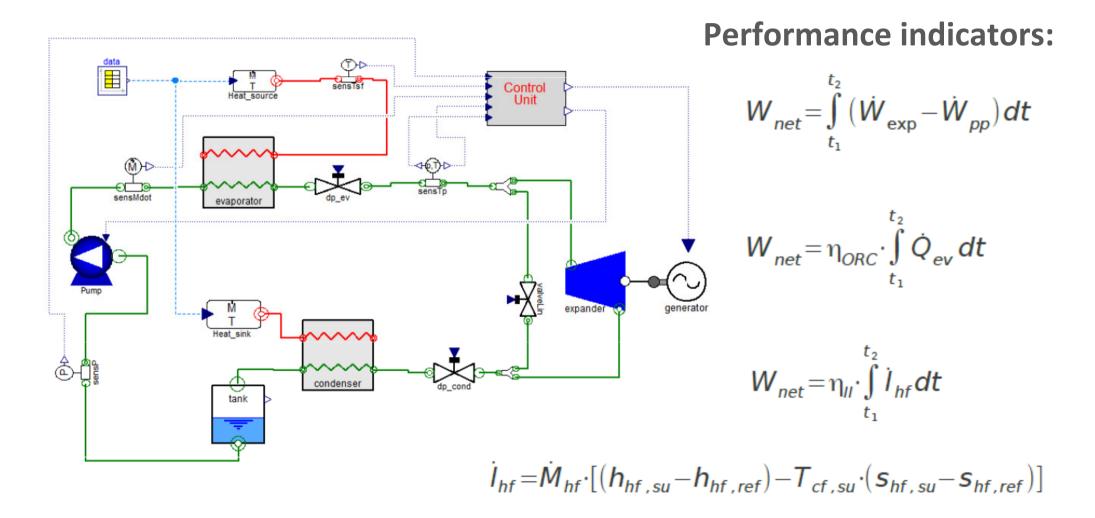
Conservation of mass:

Metal wall:

$$V_{i} \cdot \rho_{i} \cdot \frac{\partial h_{i}}{\partial t} = \dot{M}_{i-1}^{*} \cdot (h_{i-1}^{*} - h_{i}) - \dot{M}_{i}^{*} \cdot (h_{i}^{*} - h_{i}) + \dot{Q}_{i} + V_{i} \cdot \frac{dp}{dt}$$
$$\frac{dM_{i}}{dt} = V_{i} \cdot \left(\frac{\partial \rho}{\partial h} \cdot \frac{dh}{dt} + \frac{\partial \rho}{\partial p} \cdot \frac{dp}{dt}\right) = \dot{M}_{i}^{*} - M_{i-1}^{*}$$
$$c_{w} \cdot M_{w,i} \cdot \frac{dT_{w,i}}{dt} = \dot{Q}_{sf,i} - \dot{Q}_{f,i}$$



CYCLE MODEL





CYCLE DISPLAY

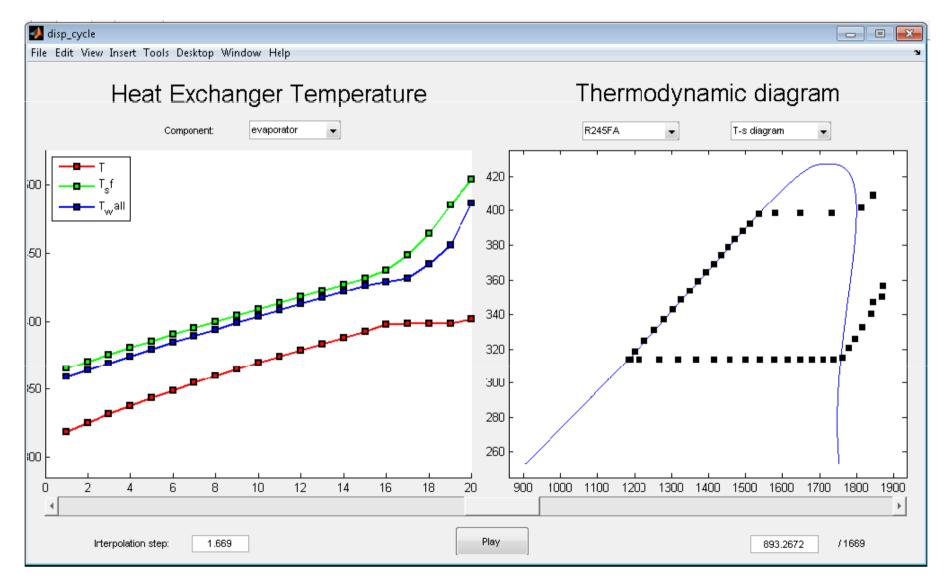




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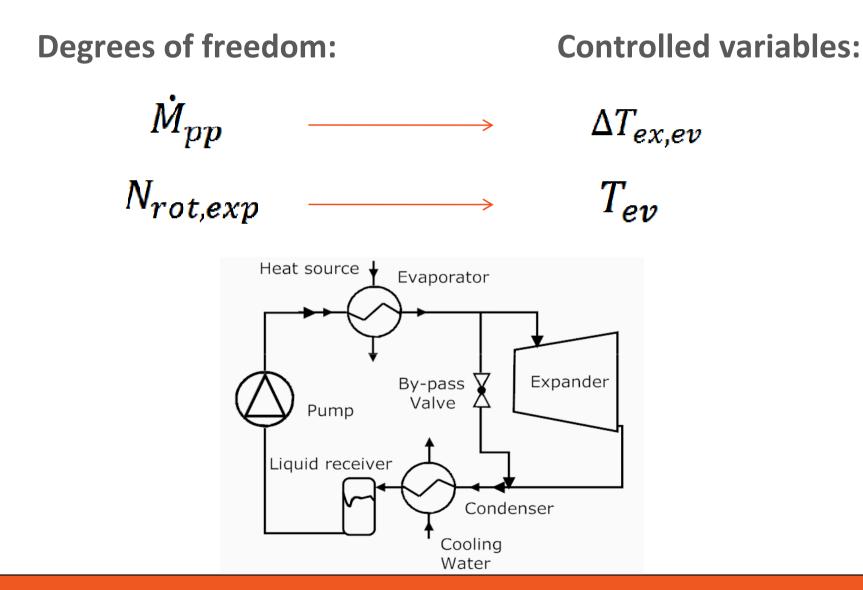
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CONTROL STRATEGIES

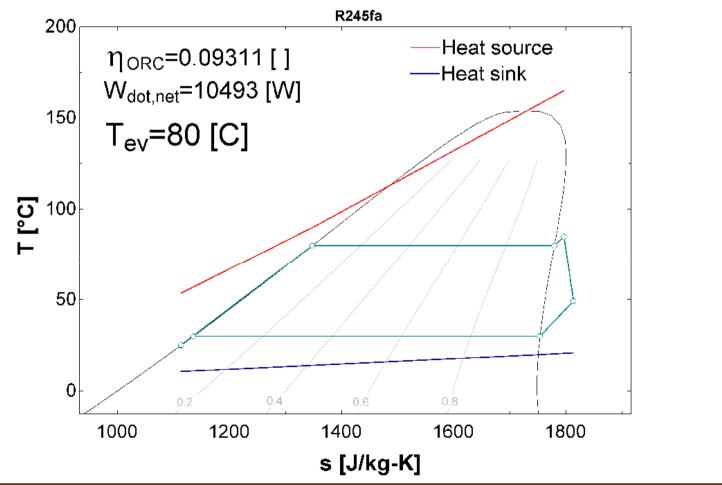


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Optimal working conditions

→ Main optimization parameter : evaporation temperature



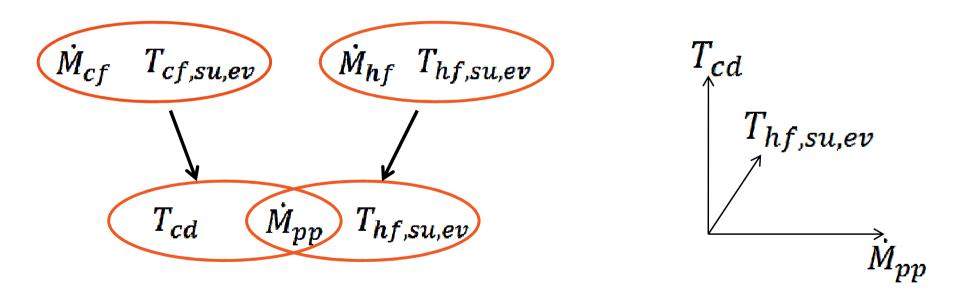




OPTIMAL EVAPORATION TEMPERATURE

Static optimization of the cycle

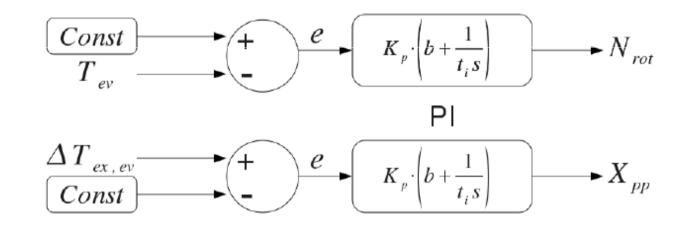
Tev, optim depends on the heat source/sink conditions:

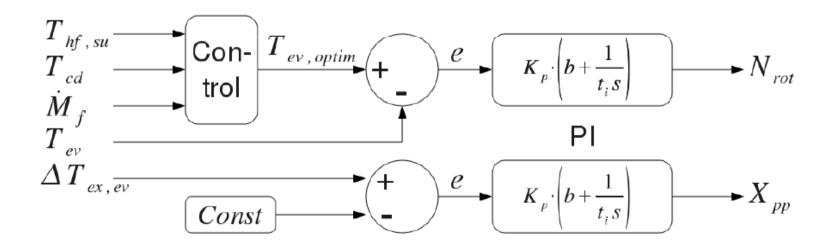


 $T_{ev,optim} = 77.6 + 4.93 \cdot 10^{-05} \cdot p_{cd} + 23.8 \cdot \ln(\dot{M}) + 7.65 \cdot \ln(T_{htf,su,ev})$ $R^{2} = 93.7\%$



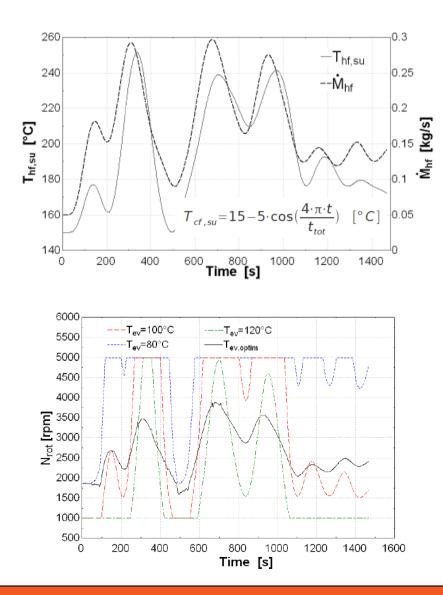
CYCLE FEEDBACK CONTROL STRATEGIES







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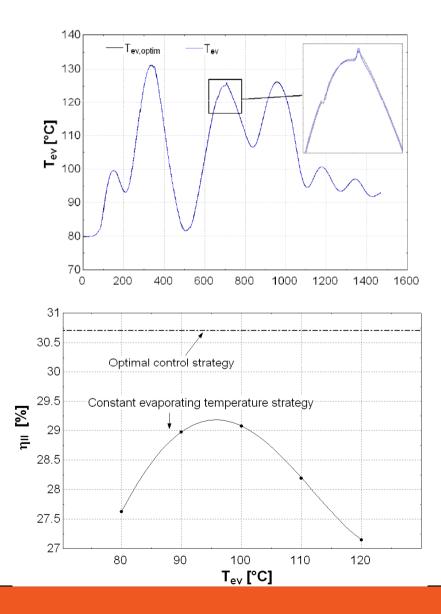




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Model predictive control

- ✓ Future action on the manipulated variable is based on a model.
- ✓ Balance between two different approaches:
 - model evaluation of the manipulated variable to reach the set point
 - Constant rectification of the trajectory by measurement of the control variable.
- ✓ Anticipates the effects of external perturbations
- ✓ Well-performing with constrained processes

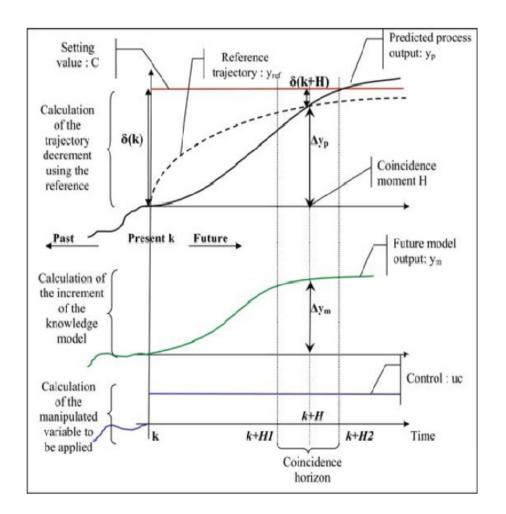


Control unit

✓ First order model to predict the process:

 $G(s) = \frac{\text{process output}}{\text{process input}} = \frac{K \cdot e^{-T_d s}}{1 + \tau \cdot s}$

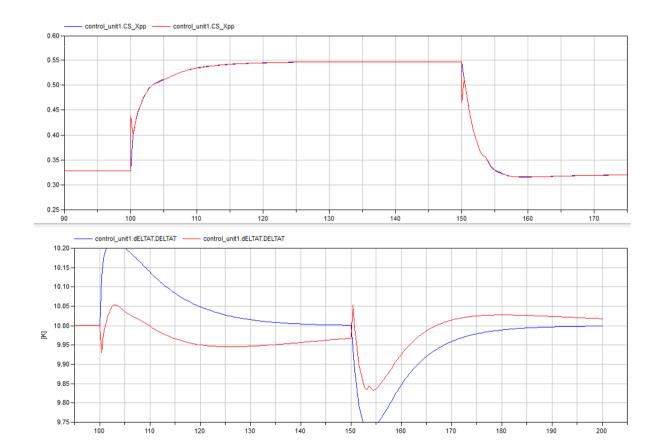
- ✓ Defined a desired trajectory to the set point
- Define an "horizon time" for which the model predicts an intersection between the desired trajectory and the process.





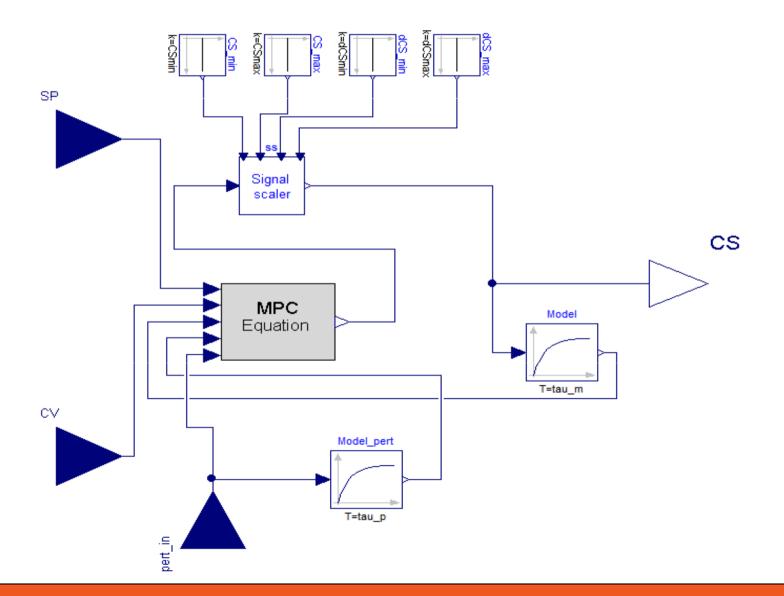
Accounting for measured pertubations

- The effect of an external pertubation is anticipated by the control.
- A first order model must be defined to account for these external influences.





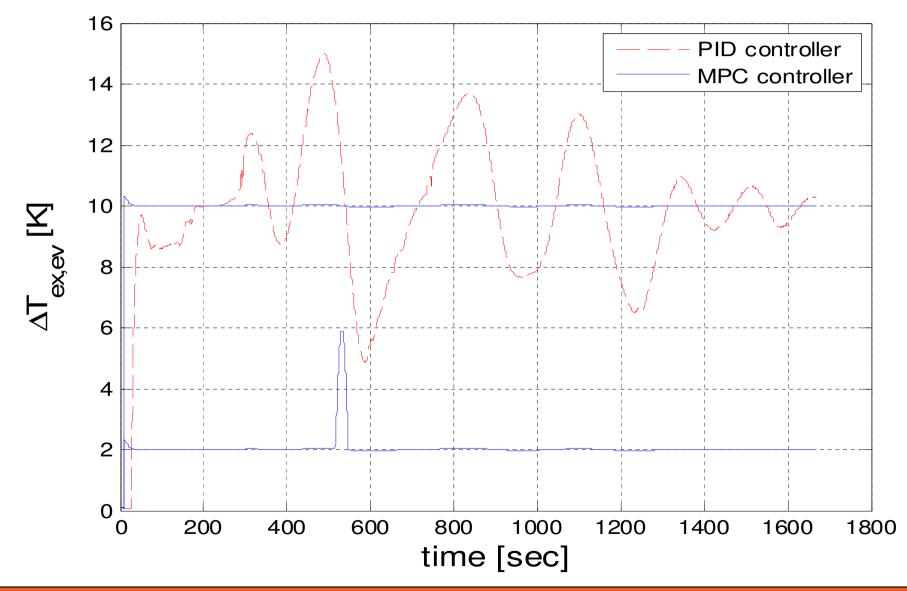
Modelica implementation



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Simulation results





Summary

- A robust dynamic model of an ORC has been developed, based on validated steady-state component models
- ✓ Different control strategies have been proposed an compared
- ✓ A varying set-point control strategy allows increasing the output by more than 4%
- ✓ MPC controller follow the superheating set point in a much better way than PI-based controllers.
- Future work will focus on the implementation and the improvement of MPC controllers with delayed processes, during start & stop procedures and in more severe conditions.



Thank you!