

# THERMODYNAMIC INVESTIGATION ON DIFFERENT ORC CONFIGURATIONS

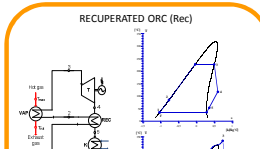
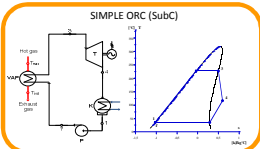
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## Abstract - aim of the work

The Organic Rankine Cycle (ORC) is an emerging technology for power generation and heat recovering from different thermal sources. The ORC adopts a simple thermodynamic concept, based on the use of organic fluids as working medium with a strong influence on the cycle performance. Common and innovative applications: biomass power plants, wasted heat recovery from industrial process and ICE flue gases, geothermal, solar desalinization and waste-to-energy power plants.

This poster represents a **PRELIMINARY NUMERICAL STUDY TO ASSESS THE RELEVANCE OF THE CYCLE LAYOUT, OF THE MAIN CYCLE PARAMETERS AND OF THE WORKING FLUID ON THE ORC PERFORMANCE**. To improve the recovery efficiency and the net power output of the SIMPLE ORC, different cycle modifications are evaluated, such as SUPERHEATED, RECUPERATED, RIGENERATIVE, SUPERCRITICAL and THEIR COMBINATIONS. A parametric analysis has been carried out at different hot source temperatures to identify the best operating condition in terms of thermodynamic and recovery efficiency, power output and volumetric expansion ratio.

## ORC thermodynamic investigation



Fluid	type	T <sub>c</sub> [°C]	p <sub>c</sub> [bar]	T <sub>lim min</sub> [°C]	T <sub>lim max</sub> [°C]
MD3M	Siloxane	355.25	9.96	6.85	400
MD2M		326.25	11.79	32.85	400
MDM		290.94	14.15	-108.15	400
benzene	Aromatic	289.01	48.98	5.85	350
toluene		318.65	41.06	12.85	400
p-xylene		343.08	35.11	34.85	330
isobutane	HC	135.05	36.50	-108.15	135
isopentane		187.25	33.90	-53.15	187
butane	Fluorinate	152.01	37.97	-91.15	400
PP2		221.85	20.60	0.00	600
PP5		291.85	17.88	0.00	400
R134a	Refrigerant	101.06	40.56	-63.15	101
R245fa		154.05	36.40	-73.15	227

ORC efficiency:  $\eta = \frac{W}{Q} = \frac{h_1 - h_2}{h_1 - h_3}$

Heat recovery factor:  $e = \frac{T_2 - T_3}{T_1 - T_3}$

Recovery efficiency:  $\eta_{re} = \frac{P}{Q_{in}} = \eta e$

Turbine specific work:  $W = \eta_{re}(h_1 - h_2)$

ORC power:  $P = \dot{m}_{in} W$

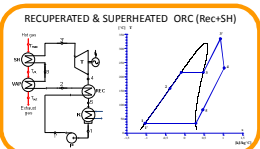
Expander volumetric ratio:  $VER = v_1 / v_2$

Overall Perf. Index:  $PI = \frac{1}{2} \left( \frac{W}{Q_{in}} + \frac{h_1 - h_2}{h_1 - h_3} + \frac{P}{P_{max}} + \frac{VER}{VER_{max}} \right)$

**Common constraints:**  
 $T_2 - T_3 = \Delta T_{min}$   
 $T_1 \leq T_{max} - \Delta T_{min}$   
 $T_1 \geq T_{min} = 35^\circ\text{C}$  or  $p_1 \geq p_{sat} = 0.03\text{bar}$

**Specific constraints for each layout configuration:**  
 e.g. for SIMPLE ORC:  
 $p_{min} = \min(p_2, p_3)$

**Degrees of freedom:**  
 evaporation temperature & pinch point position



ORC CYCLE LAYOUT

ORGANIC FLUIDS

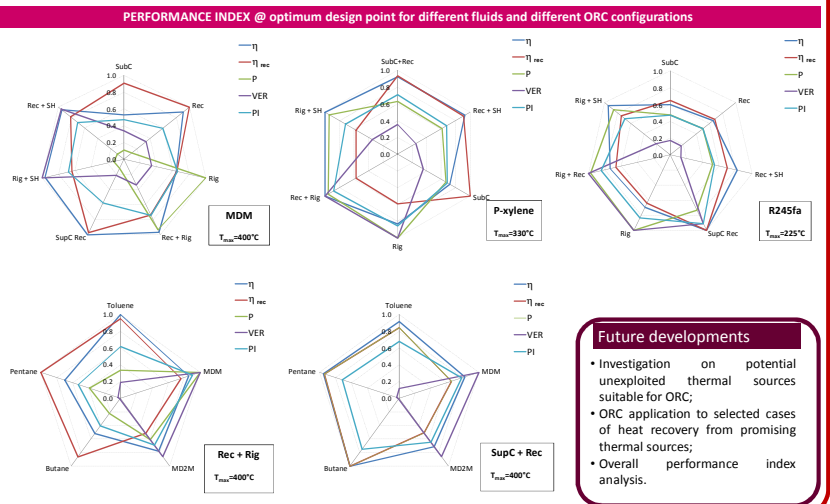
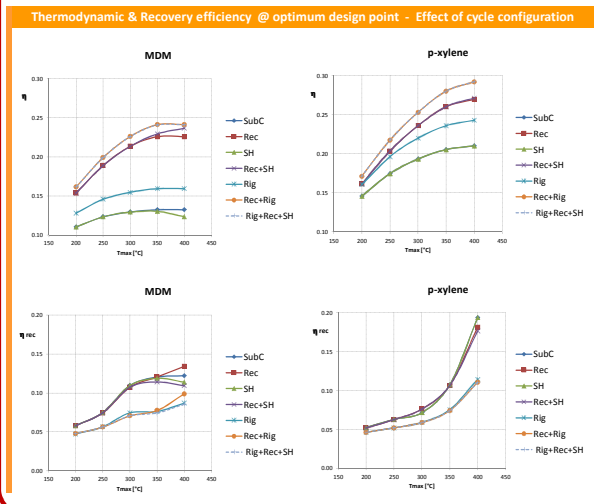
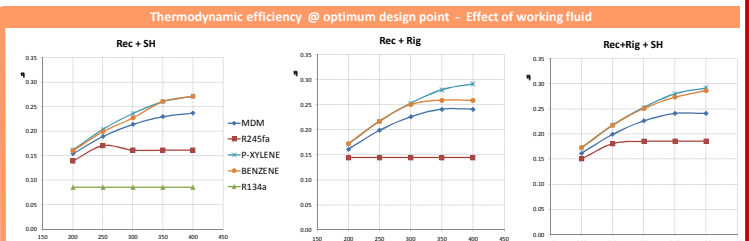
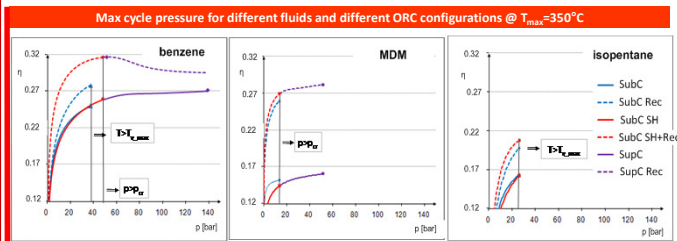
THERMODYNAMIC INPUT, CONSTRAINTS DEFINITION & PERFORMANCE INDEXES

**Common assumptions:**

T <sub>1</sub> [°C]	35
ΔT <sub>min</sub> [°C]	20
pinch [°C]	10
T <sub>max</sub> [°C]	15
η <sub>is</sub> [%/s]	1
η <sub>ex</sub> [%]	0.75
κ <sub>ex</sub> [1]	0.80
Δp [bar]	1

THERMODYNAMIC ANALYSIS & DESIGN OPTIMIZATION (evap. Temp. & pinch) with IN HOME DEVELOPED SOFTWARE BASED ON FluidProp Database

## Results overview



**Future developments**

- Investigation on potential unexploited thermal sources suitable for ORC;
- ORC application to selected cases of heat recovery from promising thermal sources;
- Overall performance index analysis.